

Forced wakes far from threshold: Stuart-Landau equation applied to experimental data

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In this Rapid Communication, we study with the Stuart-Landau (SL) amplitude equation, a wake flow control scenario using experimental data from a cylinder wake forced by plasma actuators. Given the formal framework recently discussed by Gallaire *et al.* [*Fluid Dyn. Res.* **48**, 061401 (2016)] on pushing amplitude equations far from threshold, we analyze experimental data of a forced wake in order to test the SL reduced order model. Linear stability theory and global mode concepts are used to determine the SL parameters. The extension to forced wakes of the SL model had been proposed by Thira and Wesfreid [*J. Fluid Mech.* **579**, 137 (2007)] in the context of their study on stability properties, but its employment still remained an open question. Here, we show that a forced wake at a Reynolds number far from the first threshold can also attain the critical behavior described by the SL model.

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Controlling wake flows presents many important applications in engineering and environmental sciences. The goals of control can be, for instance, the reduction of drag [1] or the modification of heat transfer on a structure subjected to an external flow [2]. Mechanically, the characteristic vortex shedding of wake flows is one of the cornerstones of many fluid-structure interaction problems, such as the vibration of bridge support cables or the thrust production mechanisms in animal locomotion. In active control strategies, considerable energy savings or engineering optimizations can be obtained through a careful choice of forcing parameters. In this Rapid Communication, we consider the canonical example of a circular cylinder, using an active control strategy based on the near-wall flow manipulation produced by plasma actuators [3]. The wake dynamics can be strongly affected when forcing the flow in the boundary layer around the cylinder.

Many studies on the subject of forced wakes have been conducted in the past 40 years using experimental, numerical, and theoretical approaches (see, e.g., Refs. [4–10]), and significant drag reductions have been associated with the stabilization of the Bénard–von Kármán instability (BvK) [6,9,11]. For wake flows with higher Reynolds numbers, or with less-energy-expensive actuators, comparable drag reductions have been attained through interfering with the formation of BvK structures by promoting symmetrical vorticity patterns [12,13].

The goal here is to examine a reduced model of the wake flow, the Stuart-Landau (SL) equation [14,15], in the case of a forced wake, identifying the parameters of the model with measurements from experiments. The application of the SL model to the unforced cylinder wake far from the

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43 threshold of the BvK instability has been discussed recently [16]. Here, we show, using plasma-
 44 actuated wake experiments, that the forced wake can also be described in the same framework,
 45 exhibiting a critical behavior coupled to the forcing parameter.

46 It is well known that the Reynolds number $\text{Re} = u_\infty d / \nu$ controls the transition from a stationary
 47 state in the wake to the BvK vortex shedding regime, where u_∞ is the flow velocity far away from
 48 the cylinder of diameter d and ν is the kinematic viscosity of the fluid. For the laminar regime in
 49 a two-dimensional (2D) flow, from $\text{Re} \sim 5$, a recirculation region with two well-defined eddies of
 50 opposite circulation takes place, and therefore the streamwise central velocity u_x is negative in the
 51 near wake—we will consider hereafter a Cartesian (x, y) reference frame with the cylinder axis z
 52 perpendicular to the (x, y) plane and placed at its origin, and the flow far away from the cylinder
 53 moving in the positive x direction. At $\text{Re}_c \simeq 47$ these eddies are no longer stable and the BvK
 54 instability sets in [4,17].

55 The wake dynamics close to the threshold can be modeled considering it as a propagating wave
 56 that grows from the origin, reaches a maximum, and decays afterwards. The spatial envelope of this
 57 coherent oscillation gives the amplitude of the so-called global mode (see, e.g., Refs. [18–20]), for
 58 which the dominant contribution is given by the first harmonic. The main flow structures, a double
 59 row of staggered vortices of opposite signs, are present for a large range of Reynolds numbers. The
 60 evolution of these structures and the supercritical transition near the stability threshold Re_c can be
 61 approximated with the complex Stuart-Landau (SL) equation (see, e.g., Refs. [4,21])

$$\tau \frac{da}{dt} = \epsilon(1 + ic_0)a - g(1 + ic_1)|a|^2 a, \quad (1)$$

62 where $a = \rho e^{i\phi}$ is a complex amplitude such that ρ is the amplitude of the global mode and ϕ
 63 the phase of the fluctuations, and τ , g , c_0 , and c_1 are real coefficients. The Reynolds-dependent
 64 control parameter ϵ is a function of the distance to the critical Re_c value and can be written as
 65 $\epsilon = \text{Re}^{-1} - \text{Re}_c^{-1}$, as proposed by Ref. [22] and thoroughly discussed by Ref. [16]. The linear term
 66 in Eq. (1) characterizes the growth rate $\sigma_0 = \epsilon$ and the frequency $\omega_0 = \epsilon c_0$ of the fluctuations in
 67 the wake at the onset, while the saturating nonlinear term $\propto a^3$ is linked to the presence of a limit
 68 cycle due to the interaction between the zeroth, or stationary, mode that represents the time-mean
 69 flow and the first harmonic of the perturbation [22]. For the limit cycle regime, this model gives the
 70 evolution of the amplitude $\rho \sim \sqrt{\epsilon/g}$ and frequency shift $\tau \dot{\phi} \sim \epsilon(c_0 - c_1)$.

71 Concerning flow control, even though forced wakes lead to complex behaviors, they can be also
 72 modeled by the SL equation. Previous attempts can be found in Refs. [4,23,24] where a temporal
 73 forcing term $F \exp(2\pi i f_f t)$ is added to Eq. (1). However, in such models there is no link between
 74 the global mode dynamics and changes produced by the mean flow correction. On the other hand,
 75 it has been suggested [11] that the selected global modes in forced wakes could exhibit a similar
 76 critical behavior as that of free wakes near the threshold Re_c . Consequently, dynamic properties such
 77 as growth rate and frequency could be described by coupling the response amplitude a to a forcing-
 78 dependent parameter. A general case can be described considering a forcing with two parameters:
 79 an amplitude A and a frequency f_f ; and the archetype of a cylinder forced by rotary oscillations
 80 has been discussed with numerical results by Ref. [25]. In such a forced cylinder wake, we also
 81 note a strong modification of the recirculation length behind the cylinder L_R , which corresponds
 82 to the streamwise location for which the streamwise time-mean flow $u_x = 0$ at $y = 0$ (see Fig. 1).
 83 If we consider the forcing as a mean flow perturbation that contributes to the zeroth harmonic
 84 of the wake, the following system for the established regime is obtained through a multiple-scale
 85 expansion [16,25],

$$\tau \dot{a} = \epsilon(1 + ic_0)a - g(1 + ic_1)|a|^2 a - \gamma(1 + i\beta)ab, \quad (2a)$$

$$b = F(|a|^2, A, f_f) = G(A, f_f) - H(A, f_f)|a|^2, \quad (2b)$$

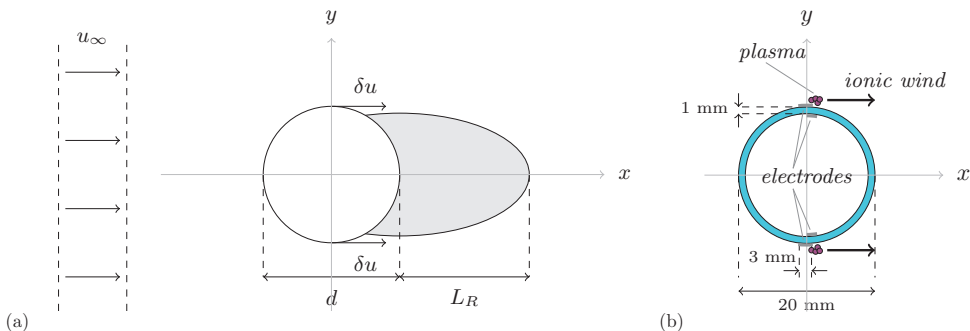


FIG. 1. Schematic diagrams of (a) a forced wake where the forcing is represented by a perturbation δu on the upper and lower parts of the cylinder that modifies the mean flow and so L_R . (b) Actual experimental setup, when forcing is obtained through plasma actuation.

86 where the dot stands for the time derivative, and γ and β are real coefficients. This leads us to define
 87 a forcing-dependent parameter b that can be represented either by the function $F(|a|^2, A, f_f)$ with
 88 A the forcing amplitude and f_f its frequency as suggested in Ref. [11], or by the pair of functions
 89 $G(A, f_f)$ and $H(A, f_f)$ as derived from the SL development. Therefore, the term $-\gamma(1 + i\beta)ab$
 90 represents the coupling between the wake dynamics and the external forcing. Writing the complex
 91 amplitude $a = \rho e^{i\phi}$, Eq. (2a) leads to

$$\rho^2 = (\epsilon - \gamma b)/g = \hat{\epsilon}/g, \quad (3a)$$

$$\tau \dot{\phi} = \epsilon(c_0 - c_1) - \gamma b(\beta - c_1). \quad (3b)$$

92 The new control parameter $\hat{\epsilon}$ in Eq. (3b) governs the supercritical transition observed in the forced
 93 wake. The $\hat{\epsilon}$ parameter depends both on the Reynolds number (through ϵ , so that it can still describe
 94 the free wake transition) and the forcing term b . This means that stabilization can occur at a Re
 95 number other than the threshold without forcing Re_c .

96 While increasing the forcing amplitude A , the perturbation amplitude ρ , and accordingly $|a|^2$,
 97 might decrease. Thus, though we do not know the exact expression of the functions F , G , and H
 98 derived in (2b), there is strong evidence that $F(|a|^2, A, f_f) \rightarrow G(A, f_f)$ at high values of forcing,
 99 so that the parameter b only depends on the forcing properties A and f_f .

100 As mentioned above, the objective of the present Rapid Communication is to apply this
 101 theoretical framework to a realistic experimental case. For that purpose we use the experimental
 102 study of forced flow past a circular cylinder described in Ref. [26]. The setup, where plasma
 103 actuators are used to induce a ionic wind on the boundary layer of the cylinder, is represented
 104 schematically in Fig. 1. The cylinder of diameter $d = 0.02$ m is placed in an air flow of upstream
 105 velocity $u_\infty = 0.18$ m/s with viscosity $\nu = 1.5 \times 10^{-5}$ m²/s, which gives a Reynolds number
 106 $Re = 235$.

107 Forcing over the cylinder surface produces a strong coherent flow along the cylinder span which
 108 diminishes the possible spanwise variations due to three-dimensional instabilities or end effects
 109 [8,27,28]. Flow field measurements are performed using 2D particle image velocimetry (PIV)
 110 [29]. We refer the reader to D'Adamo *et al.* [26] for more details on the experimental setup. The
 111 experimental forcing parameter dc stands for duty cycle, a relationship between the time when the
 112 plasma is on, T_{ehd} , and the forcing time T_{burst} (see Ref. [29]). It is thus related with the actuator
 113 electric energy input (see Ref. [26]). As suggested in earlier studies, the modification of the flow
 114 transition could be forced by changing the actuation amplitude, its frequency, or both [11]. Here, it is
 115 the ionic wind amplitude of the plasma actuators that sets the amplitude of the forcing perturbation
 116 δu , showed schematically in Fig. 1. The forcing is stationary so it has no time dependence ($f_f = 0$)
 117 and, while we do not have a direct measurement of the velocity perturbation $\delta u \sim A$ added in

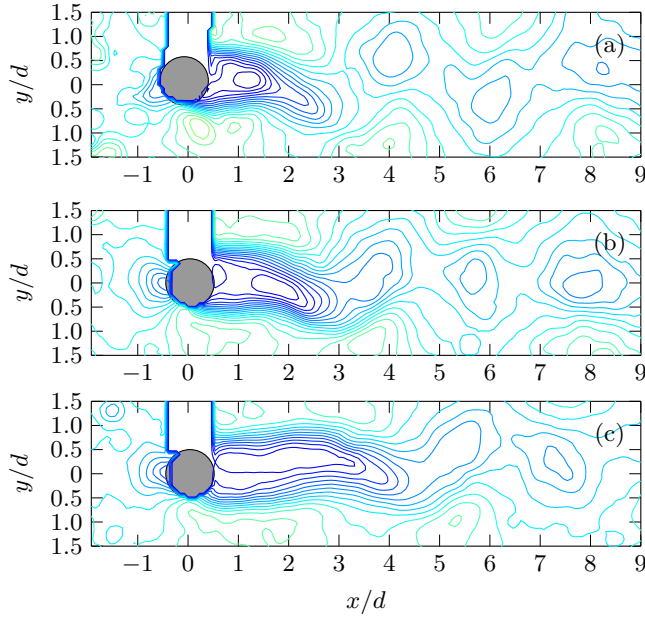


FIG. 2. Isocontours of the instantaneous 2D velocity modulus of the wake behind the cylinder for different duty cycles. (a) $dc = 0$ (no forcing), (b) $dc = 10$, (c) $dc = 20$. While increasing the forcing amplitude, an extension of the recirculation region length L_R , and an increase of the wavelength are observed.

118 the neighborhood of the cylinder wall (see Fig. 1), we know it is an increasing function of dc .
 119 We thus keep the nomenclature of the experimental setup forcing parameter dc in the following
 120 discussion.

121 Figure 2 shows typical wakes for different forcing amplitudes: $dc = 0$ (no forcing), $dc = 10$,
 122 and $dc = 20$. The velocity modulus $|\mathbf{u}| = (u_x^2 + u_y^2)^{1/2}$ is represented, where $\mathbf{u} = (u_x, u_y)$ is the
 123 velocity field measured experimentally. As we increase the forcing amplitude, we observe both a
 124 growth of the recirculation region and a stabilization of the wake, as vortex shedding takes place
 125 further downstream. Such changes are linked to the mean flow correction [8,30] and in this case
 126 they are introduced by the forcing. In order to examine the modification of the wake induced by the
 127 forcing, we explore now the evolution of the mean flow velocity. Figure 3(a) shows the evolution
 128 of the time-mean streamwise velocity $\langle u_x \rangle$ measured along the x axis at $y = 0$ as a function of the
 129 intensity of the plasma forcing dc . In the same graph, we observe that the mean velocity difference
 130 Δu_{\min} between the natural wake and the forced one increases with dc , and it can be noticed that both
 131 the minimal value of $\langle u_x \rangle$ and its distance from the cylinder are modified. As dc increases, the mean
 132 flow correction has smaller variations so its global contribution to the zeroth harmonic is decreasing,
 133 which means that the wake tends to be stabilized. Now, the amplitude ρ of the SL-equation limit
 134 cycle can be estimated from intensity of the flow fluctuations. Following Ref. [30], we identify the
 135 order parameter ρ^2 from Eq. (3a) with the maximum of the transverse velocity fluctuations $\langle u_y'^2 \rangle^{1/2}$
 136 along x at $y = 0$ [see Fig. 3(b)]. The position of this maximum changes with the strength of the
 137 forcing, as was also reported in Thiria and Wesfreid [25].

138 Let us recall the study of Zielinska *et al.* [19] on the mean flow modification of a wake in the
 139 supercritical regime. In their notation, when the flow is no longer stationary, the time-mean flow
 140 $\langle V \rangle = V_0 - \delta V$ results from a nonlinear correction δV to the stationary basic flow V_0 . In the present
 141 case, the mean flow modifications represented by the quantity Δu_{\min} are portrayed with respect to
 142 the nonlinear oscillations represented by ρ^2 , as shown in Fig. 3(c). When the forcing is strong
 143 enough, $\Delta u_{\min} > 0.1$, ρ^2 depends linearly on the modifications of the mean flow Δu_{\min} . Thus,

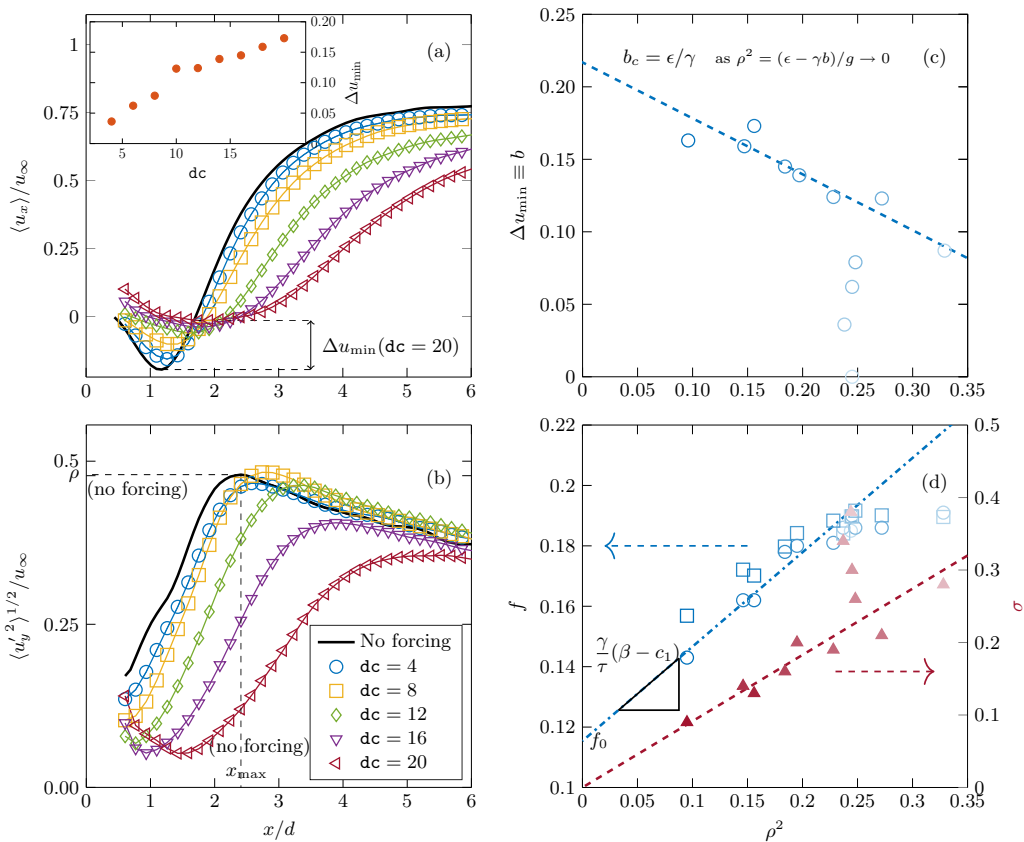


FIG. 3. (a) Evolution of the mean velocity measured along the x axis as a function of dc , where the different labels are displayed on (b). The definition of Δu_{\min} is plotted in the case of $dc = 20$. The inset plot shows the values of Δu_{\min} as a function of the forcing parameter dc . (b) Evolution of the spatial shapes of the global modes with the forcing dc . For each curve, its maximum defines a pair of values (x_{\max}, ρ) . (c) Relationship between the squared amplitude ρ^2 and the characteristic modification of the mean flow given by Δu_{\min} . As Δu_{\min} increases, ρ^2 tends to 0, corresponding to the wake stabilization. Linearity ceases to hold when the forcing weakens, which is displayed in the graph through a diminished color intensity of the plot markers. (d) Behavior of the selected frequency f of the global mode (\circ and \square) and the growth rate σ (\blacktriangle) as a function of the squared amplitude ρ^2 ; \circ and \blacktriangle are obtained through a linear stability analysis, and \square are determined from a Fourier decomposition of the experimental values. As $\rho^2 \rightarrow 0$, the stable mean flow frequency $f_0 \sim 0.115$. In the same sense, while increasing the forcing, the temporal growth rate decreases.

144 following Eq. (2a), we propose that the parameter for the mean flow correction b is equivalent to
 145 Δu_{\min} . The slope of the line displayed in Fig. 3(c) is therefore $-\gamma/g$.

146 To explore the stabilization of the wake, we perform a linear stability analysis based on the
 147 time-mean flow, as discussed by Refs. [31,32]. Following the methods provided for a local analysis
 148 by Ref. [33], we have access to the growth rate σ_0 and the frequency ω_0 of the selected unstable
 149 mode for each x -streamwise coordinate. The criterion for the prediction of global frequency for
 150 a spatially developing flow [34] is recalled in Appendix B. This predicted value is compared to a
 151 direct measurement through a Fourier decomposition of the experimental values $u_y(t)$ evaluated at
 152 a vicinity of $(x_{\max}, y = 0, t)$. They are plotted in Fig. 3(d), showing that the linear stability analysis
 153 is accurate enough in our case. As we can see, the forced wakes tend to a stable stationary solution
 154 as the growth rate σ is a decreasing function of the nonlinear oscillations given by ρ^2 . The behavior
 155 of the wake properties σ and f is very similar to the evolution observed for free wakes when

156 approaching the threshold of the instability by upper values of the Reynolds number ($\text{Re} \rightarrow \text{Re}_c^+$).
 157 It is noticeable in Eq. (3a) that when ρ^2 goes towards zero, which corresponds to $\hat{\epsilon} = 0$, we
 158 obtain a critical value $b_c = -\epsilon/\gamma$ that we can use to create a nondimensional transition parameter
 159 $(b_c - b)/b_c$. In the same sense, for small values of ρ , the frequency of the wake decreases through
 160 a linear dependence of slope $\gamma(\beta - c_1)/\tau$. Particularly, when $\rho^2 \rightarrow 0$, the forced wake frequency
 161 reaches a finite value identified as the stable flow frequency $f_0 \simeq 0.12$ [18,33], as in the nonforced
 162 scenario [see Fig. 3(d)]. The wave velocity $c = \lambda f$ is near constant ($c = 0.80 \pm 0.05$). The reduced
 163 parameter $(b_c - b)/b_c$ is therefore relevant to describe the forced wake transition at a fixed
 164 Reynolds number Re , and the parameter $\hat{\epsilon}$ would be a good choice to fully describe the Reynolds-
 165 dependent and forcing-dependent transition that leads to the BvK instability when we work in flow
 166 control.

167 Summarizing, we found strong evidence of a critical behavior of the mean flow parameter $b \rightarrow b_c$
 168 at the transition to the flow stabilization. Consequently, the mean flow correction produced by the
 169 control is found to be a parameter for the bifurcation in the way defined in Eq. (2). The estimated
 170 Strouhal number $\text{St} = fd/u_\infty$ corresponding to the value $b = b_c$ was $\text{St} \simeq 0.12$, which is only
 171 slightly below the critical frequency selected at $\text{Re} = \text{Re}_c$.

172 Scaling laws based on the mean flow modification seem to remarkably describe and characterize
 173 the critical dynamics close to threshold under forcing conditions when flow control is performed
 174 with plasma actuators. Even more, the mean flow, which is a function of the forcing parameter
 175 (A, f_f) and the Reynolds number, does not involve directly the specific form of the external
 176 perturbation, but its consequence on average. The scenario should thus be similar with other forms
 177 of control such as body rotatory oscillations (see Ref. [35]).

178 The link between the mean flow correction Δu_{\min} and the transition in the forced wake
 179 being established, we believe that this parameter, which is closely linked to the stabilization
 180 mechanisms, should be robust enough to describe critical behaviors in wake flows for a large
 181 range of forcing parameters and far from the critical Reynolds number. As a final note, we
 182 recall that, as the forcing also modifies the recirculation length scale L_R , a transition diagram
 183 between stable and unstable flow had been suggested in the $(\text{Re}; L_R)$ plane (see Fig. 24 in
 184 Ref. [11]). The behavior of the recirculation length, both for the stable state and the vortex shedding,
 185 could be explained as a linear and nonlinear modification of the mean flow [19,30]. Therefore,
 186 we can relate it to the perturbation induced by the forcing and help to complete a transition
 187 diagram.

188 In this Rapid Communication, we expose that is possible to explain and predict the effect of an
 189 actuator on unstable flows. The steps described here offer a physical explanation of flow control
 190 mechanisms that go beyond a purely phenomenological description.

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 193 of Fluids).

194 APPENDIX A: STUART-LANDAU EQUATION WITH A FORCED TERM

195 We develop the equation corresponding to the forcing cases from the basis of the Stuart-Landau
 196 equation applied in wake flows by Refs. [16,22],

$$\frac{dA}{dT} = \lambda A - \mu |A|^2 A, \quad (\text{A1a})$$

$$\lambda = \lambda_0 + \delta\lambda(F), \quad (\text{A1b})$$

$$\mu = \mu_0 + \delta\mu(F), \quad (\text{A1c})$$

197 where λ_0 and μ_0 are the coefficients when there is no forcing and $\delta\lambda(F)$ and $\delta\mu(F)$ are functions
 198 that depends on forcing. Therefore, Eq. (A1a) can be rewritten as

$$\frac{dA}{dT} = \lambda_0 A - \mu_0 |A|^2 A + \beta B A, \quad (\text{A2a})$$

$$0 = -\beta B + f(|A|^2, F), \quad (\text{A2b})$$

$$f(|A|^2, F) = \delta\lambda(F) - \delta\mu(F)|A|^2, \quad (\text{A2c})$$

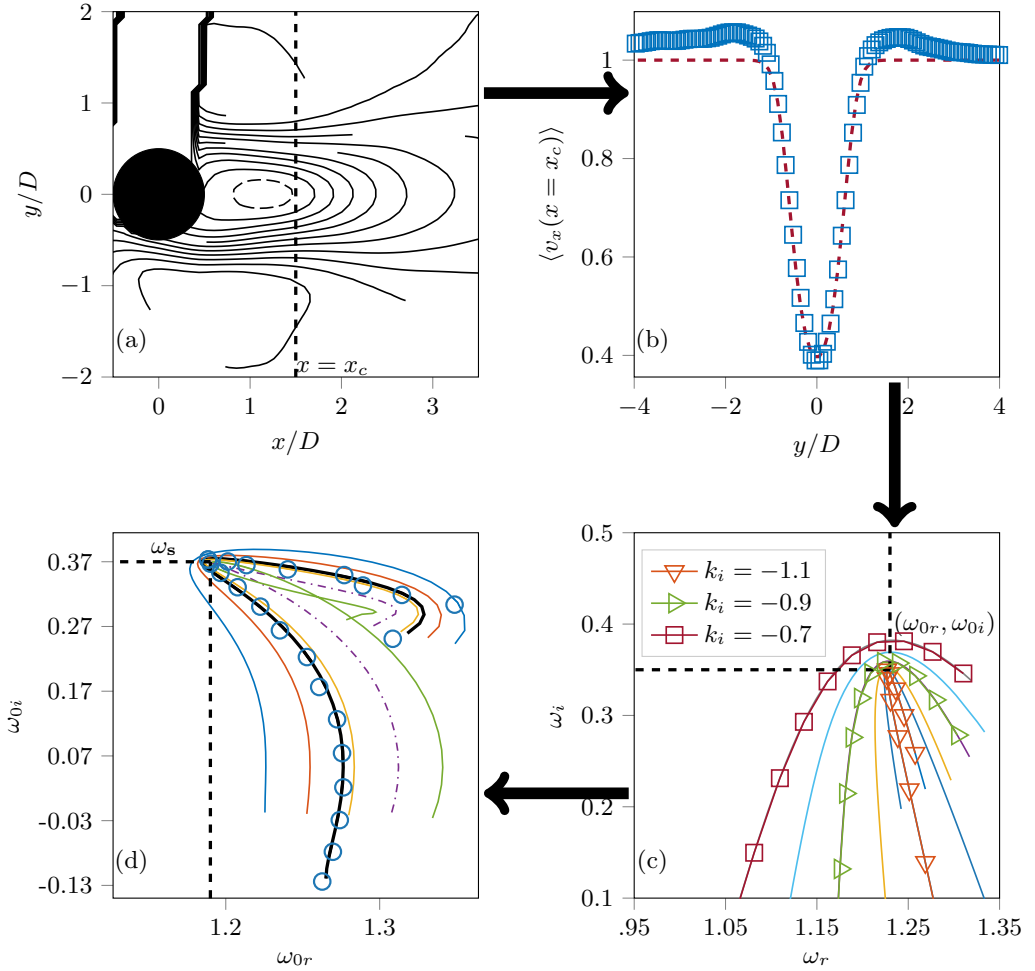


FIG. 4. Schematic representation of the necessary steps involved in the linear stability analysis: (a) Mean flow streamwise velocity contours, with a slight loss of symmetry is due to forcing, where a cut a $x = x_c$ gives (b) a mean velocity profile (square symbols) $\langle v_x(y, x = x_c) \rangle$, which is fitted by a function (dashed line) $u(y) = 1 - a_0 + a_0 \tanh [a_1(y/d)^2 - a_2]$, where a_0 , a_1 , and a_2 are fitting parameters. Solving the corresponding dispersion relation, the Rayleigh equation for this inviscid instability, for a range of wave numbers $k = k_r + ik_i$, we obtain (c) a map for $k_i = \text{const}$ in the ω plane. At a critical point, a cusp on the $k_i = -0.7$ curve, the point $(\omega_{0r}, \omega_{0i})$ is determined for x_c . In (d) the results for each x position (o marks) are plotted in a ω plane. Criteria for the prediction of a global frequency in a spatially developing flow is given by the condition $\partial\omega/\partial x|_{x=x_s} = 0$ [34]. The saddle point $\omega_s = \omega(x = x_s)$ is obtained through an analytic continuation to complex values of $x = x_r + ix_i$ by inspecting the when a cusp point takes place (point-dashed curve) [36]. Considering all the cases, the points $\omega_s = (\omega, \sigma)$ provide the data for Fig. 3(d).

199 which is a particular case for Eq. (2), corresponding to the stationary evolution of B . A possible
 200 modification for an harmonic forcing is to consider $f(|A|^2, F)e^{i\omega_F t}$, as proposed by Ref. [11].

201 APPENDIX B: PREDICTION OF THE GLOBAL MODE FREQUENCY

202 The criterion proposed by Ref. [34] and applied to a wake flow in Refs. [36,37] for the
 203 prediction of global frequency for a spatially developing flow consists in finding a saddle point
 204 $\partial\omega_0/\partial x|_{x=x_s} = 0$ of the complex function $\omega_0(x)$ through use of the Cauchy-Riemann equations
 205 and analytic continuation to complex values of $x = x_r + ix_i$. The procedure, applied as in other
 206 works [11,38] to the time-mean flow, is full summarized in Fig. 4. Hence, we could select the onset
 207 frequency $f = \omega_0(x_s)/2\pi$ for this spatially developing flow.

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